

ORDER-NO: 5872176002

# ZF - REVERSING GEAR 

## 6 WG-200

## DESCRIPTION

INSTALLATION INSTRUCTIONS OPERATION
MAINTENANCE

## TECHNICAL DATA

| Engine power : | Max. KW | 220* |
| :---: | :---: | :---: |
| Turbine moment: | Max. Nm | 1450 * |
| Engine speed : | Max. $\min ^{-4}$ | 2500* |
| Mass (without oil): | kg | ca. 500 to $900^{* *}$ |

## Description:

The ZF-Hydromedia Transmission 6 WG-200 is composed of a hydrodynamic torque converter and a rear-mounted multi-speed Powershift reversing transmission with integrated transfer case - see Table 1 and 2 -

The torque converter is a wear-free start-up device, wich is adapting itself infinitely variable to the required situations (required driving torque).

A lockup clutch, installed in the converter, realizes after the starting phase a mechanical connection between engine and countershaft gear.

Drive by direct mounting on the engine, through membrane for engine connection SAE 1, 2 and 3, or separate installation (drive through universal shaft) with DIN, Mechanic or Spicer drive flange.

The signals for the gear shiftings are issued, according to the design of the vehicle by the following ZF-Control units:

- Power-shift
- Semi-automatic control

EST-17 T (24 V),

- Semi-automatic control
- Fully-automatic control

EST-19 T (12 V),

- Fully-automatic control

EST-2 resp. the succeeding generations

- Fully-automatic control

EST-25 (24 V),
EST-26 (12 V).

## Torque converter:

## Different Versions:

Unit Size W 320 to W 350 with torque increase according to the Version (starting conversion 1,5 to 2,5 ) and the converter lockup clutch (WK).

All Converter Versions can be optionally equipped with a stator free wheel or resp. and with a sprag pack.

## Powershift - Reversing transmission:

6 Forward speeds and 3 Reverse speeds.

## Output:

Through integrated transfer case with following center distances of 400,500 and 555 mm .

## Power take-off:

Up to 2 engine-dependent power take-offs $\mathrm{i}=1,0$ for hydraulic pumps, optionally also disengageable.

Pump connections SAE B and C.

## Transmission accessories:

Electric or mechanical Speedometer sensor, Axle declutch, Emergency steering pump, Differential, Retarder Crawler speed Axle drive APG-8/HK or DK-7 etc. upon request.

## Transmission ratio (mechanical)*

6-Speed
1st-Speed Forward ..... 6,069
2nd-Speed Forward ..... 3,934
3rd-Speed Forward ..... 2,630
4th-Speed Forward ..... 1,705
5th-Speed Forward ..... 1,194
6th-Speed Forward ..... 0,774
1st-Speed Reverse ..... 6,069
2nd-Speed Reverse ..... 2,630
3rd-Speed Reverse ..... 1,194

[^0]
## IMPORTANT INFORMATIONS

Carry out transmission oil-level check at engine idle run (about $1000 \mathrm{~min}^{-1}$ ) and at operating temperature of the transmission.

For temperatures of about $40^{\circ} \mathrm{C}$, the lower mark on the oil dipstick is valid.
For temperatures of about $80^{\circ} \mathrm{C}$, the upper mark on the oil dipstick is valid.
ATTENTION: At standing engine, the oil level in the transmission can rise essentially, according to the installation conditions!

Respect the oil change intervals with simultaneous filter cleaning resp. filter replacement.

At the engine start, place the Controller always to the Neutral position.
Release the parking brake each time before driving off.
At the reversing, reduce the engine speed, reverse in the 1st Speed only.
Avoid to skip speeds during the drive.
The engaged speed has no effect at stopped engine.
Before leaving the vehicle, secure it additionally by brake blocks.
The towing speed must under no condition be higher than $10 \mathrm{~km} / \mathrm{h}$, the towing distance not longer than 10 km (without secondary pump).

Operating temperature $80^{\circ} \mathrm{C}-100^{\circ} \mathrm{C}$, a short-time increase up to max. $120^{\circ} \mathrm{C}$ is permitted.

Pay attention to the control pressure.
In case of irregularities on the transmission, immobilize the vehicle, ask for trained personnel.

## I. DESCRIPTION

### 1.1 Function of the converter:

The converter works according to the Trilok-System, i.e. it assumes at a high turbine speed the characteristics, and with it the favourable efficiency of a fluid clutch.
The converter is designed according to the engine power so that the most favourable operating conditions for each installation purpose are existing.

$$
\text { The torque converter is composed of } 3 \text { main components: }
$$

## Pump wheel - Turbine wheel - Stator (Reaction member)

These 3 impeller wheels are arranged in such a ring-shape system that the fluid is flowing through the circuit components in the indicated order.

The ZF-Converter and control pump is constantly pumping pressure oil through the converter. In this way, the converter can fulfill ist task, to multiply the torque of the engine, and at the same time, the heat created in the converter is eleminated by the escaping oil.
The oil escaping out of the pump wheel, enters the turbine wheel and is there inversed in the direction of flow.
According to the rate of inversion, the turbine wheel and with it also the output shaft receives a more orless high reaction moment. The stator (reaction member), following the turbine, has the task to inverse the oil, which is streaming out of the turbine, again and to deliver it under the suitable discharge direction to the pump wheel.
Due to the inversion, the stator receives a reaction moment.
The relation turbine moment/pump moment is called torque conversion. This is the higher, the greater the speed difference of pump wheel and turbine wheel will be.

Therefore, the max conversion is created at standing turbine wheel.
With increasing output speed, the torque conversion is decreasing. The adaptation of the output speed to a certain required output moment is infinitely variable and automatically achieved by the torque converter.

If the turbine speed is reaching about $80 \%$ of the pump speed, the conversion becomes 1,0 , i.e. the turbine moment becomes equal to that of the pump moment.
From this point on, the converter is working similar to a fluidclutch.
A stator free wheel serves to improve the efficiency in the upper driving range, in the conversion range, it backs up the moment upon the housing and is released in the coupling range. In this way, the stator can rotate freely.

An installed sprag pack has the purpose to connect at overrunning vehicle the drive shaft and turbine shaft (converter output) frictionally with each other.
This condition appears for example at downhill driving. In this case, the braking effect of the engine can be utilized.

The converter lockup clutch (WK), if installed, will be automatically closed at the point at which - at increasing driving speed - the conversion has dropped to about 1 , and the further driving with the open converter brings no more traction force increase.

At closed converter lockup clutch (WK) the slip between pump wheel and turbine wheel, and with it the hydraulic loss in the converter is equal to ,Zero".


### 1.2 Powershift transmission:

The multi-speed reversing transmission in layshaft design is engageable under load by hydraulically controlled multi-disk clutches.
All gear wheels are constantly meshed and running in antifriction bearings.
The gear wheels, bearings and clutches are lubricated with cooled oil.
The 6-Speed Reversing transmission is equipped with 6 multi-disk clutches.
At the shifting, the corresponding plate pack is compressed by a piston, movable in axial direction, which is pressurized with pressure oil.
A compression spring is pushing back the piston,thus releasing the plate pack. As to the Layout of the transmission as well as the indication of the closed clutches in the single speeds, see Table 1, 2, 16, 17, 18 and 19.

### 1.3 Transmission control:

Transmission control see Oil circulation diagram, Table 20,21, 22 and 23.
The converter charge and control pump, necessary for the oil supply of the converter and the transmission control, is located in the transmission upon the engine-dependent drive shaft.
The feed rate of the pump is $Q=80 \mathrm{~V} / \mathrm{min}$, at $n_{\text {Ergine }}=2000 \mathrm{~min}^{-1}$.
This pump is sucking the oil through the coarse filter out of the oil sump - the filter can be also fitted externally from the transmission - and delivers the oil through the ZF-Fine filter - this one is externally fitted from the transmission - to the control pressure valve.

Grade of filtration $\quad B_{30}{ }^{220 \mu \mathrm{~mm}} \quad B_{10}{ }^{215} \mu \mathrm{~mm}$
Filter area

$$
3500 \mathrm{~cm}^{2}
$$

The pressure oil is delivered by the pump to the control pressure valve.

## A - STANDARD - VERSION

see Oil circulation diagram Table 21 and 23 as well as Aluminum die-casting shift control Table 25.

The oil, pressurized by the control pressure valve, is delivered through the pressure control valve to the shift valves $1,2,3,4$ and 5 of the multi-disk clutches.
During the shifting procedure, the pressure control valve regulates the pressure build-up in the clutches.
At the change-over, the pressure is dropping for a short time and is increasing again to $16{ }^{+2}$ bar after the completed shifting operation.

The Signals for the gear shiftings are issued, according to the Version of the vehicle, by the following ZF-control units:

- Power-shift
- Semi-automatic control
- Semi-automatic control
- Fully-automatic control
- Fully-automatic control
- Fully-automatic control
- see Table 26, 27 and 28

EST-17 T ( 24 V ), - see Table 29, 30, 31 and 32
EST-19 T (12 V),
EST-2 resp. the succeeding generations
EST-25 (24 V),
EST-26 (12 V).

The reset valve resets the pressure control valve to its intial position, as soon as the speed has been engaged, so that the spool is ready for the next gear change.

## B - VERSION WITH VARIOVALVE

see Oil circulation diagram Table 20 and 22 as well as Aluminum die-casting shift control Page 24.

The Oil, pressurized by the control pressure valve, is directed through the 2 -stage pressure control valve with bleeder valve and variovalve to the shift valves $1,2,3,4$ and 5 of the multidisk clutches.
The pressure control valve regulates during the shifting procedure the pressure build-up in the clutches. At the change-over, the pressure is dropping for a short time and is increasing again to $16^{+2}$ bar after the completed shifting operation.

The pressure control can be „Soft "or „Fast",namely as follows:
The Signals for the gear shiftings are issued, according to the Version of the vehicle, by the following ZF-Control units:

- Power-shift
- Semi-automatic control
- Semi-automatic control
- Fully-automatic control
- Fully-automatic control
- Fully-automatic control
- see Table 26, 27 and 28

EST-17 T (24 V),

- see Table 29, 30, 31 and 32

EST-19 T (12 V),
EST-2 resp, the succeeding generations
EST-25 ( 24 V ),
EST-26 (26 V).

According to the vehicle version, the variovalve (M7) can be gear-dependently activated, thus creating 2 different pressure control curves.

At shiftings at which the respective variovalve (M7) is activated, the pressure control begins at a low starting pressure.
The 2 -stage valve is regulated, through the pump pressure, by the variovalve (M7) to the stop. This is leading to a "Soft shifting".

## Fast Shifting*

At shiftings at which the variovalve (M7) is not activated, the pressure control begins at a higher starting pressure.
The 2 -stage valve is pushing the pressurizing spool of the pressure control valve to the left. This is leading to a "Fast Shifting".

The bleeder valve has the task to ventilate the pressure chamber behind the pressure control valve for a longer time.

The reset valve resets the pressure control valve into its initial position, as soon as the speed has been engaged, so that the spool is ready for the next shifting.

These shifting operations are exactly coordinated with the different models and types of vehicle.

## From here on, the Description is applicable again for both Versions:

The pilot pressure for the control of the shift valves is reduced by the reducing valve to 10 bar and is delivered to them through the solenoid valves M1, M2, M3, M4 and M5.
The control pressure valve is limiting the max. control pressure and releases the main stream to the converter and lubricating circuit.
A safety valve is installed in the inlet to the converter, which protects the converter against high internal pressures (Opening pressure $8,5 \mathrm{bar}$ ).
Within the converter, the oil serves as power transmission according to the well-known hydrody namic principle (see Chapter Torque converter).

To avoid cavitation, the converter must be always completely filled with oil. This is achieved by a converter pressure holding valve, rear-mounted to the converter with an opening pressure of about 2,5 bar.
The oil which is escaping from the converter is directed to a heat exchanger (oil-water resp. oilair).

At warter-cooled engines, the heat exchanger has to be arranged in the cooling-water stream which is delivered to the engine.
An oil-air cooler can be also used. At air-cooled engines, the heat exchanger mut be arranged in the cold-air stream of the cooling fan.
The heat exchanger has to be connected according to the counterflow principle.
These units are not delivered by the Zahnradfabrik Passau GmbH.
For the safety of the oil cooler, a bypass valve has to be planned which is reacting in case of cold oil or a clogged oil cooler.

From the oil cooler, the oil is delivered to the transmission and from there to the lubricating-oil circuit so that all lubricating points are supplied with cooled lubricating oil.

Five (5) solenoid valves are installed in the control unit - see Table 24 and 25.
The allocation of the solenoid valves - shifting positions - to the single speeds can be taken from the Tables 16, 17, 18, 19, 20, 21, 22 and 23.

The selection of the magnets is differently combined, according to the gear,
These combinations are called coding.
The electric shift control offers the advantage of an easy and exact operation.
It is an essential advantage that there are no installation problems because of the electric cable connection.
At Converter Versions with converter lockup clutch (WK), the clutch will be automaticaly engaged.
The opening and closing of the WK is realized through a solenoid shift valve. The speed is picked up on the turbine side by an inductive transmitter.

### 1.4 Electronic control units for ZF-Powershift transmissions:

## General

Because of the different design of the electronic transmission control units at the different vehicles, the corresponding documentation must be taken from the Operating Instructions of the Vehicle Manufacturer or from the Technical Fly leafs of the relative Parts-List Versions.
In theses, the corresponding wiring and connection diagrams are laid down, (see Samples, Tables $26,27,28,29,30,31,32$ und 33).
These Informations can be requested from ZF-Passiu.
Besides, a separate Description of theses Control units is existing.
According to the type of vehicle, the wiring is executed on the base of the wiring digrams.
Variations as double-control platform operation (e.g. RT-Crane-Vehicles) are available.
The corresponding electric wiring diagrams are available upon request.
Upon request, the wiring can be also delivered by ZF :
If the wiring is executed by the Vehicle Manufacturer, it should comply with our conditions.

The Powershift transmissions are mostly equipped with the following ZF-Control units, developped for them:

- Power-shift
- Semi-automatic control
- Semi-automatic control
- Fully-automatic control
- Fully-automatic control
- Fully-automatic control

EST-17 T (24 V),
EST-19 T (12 V),
EST-2 resp, the succeeding generations
EST-25 ( 24 V ),
EST-26 (12 V).

### 1.4.1 Power-shift:

see Sample, Circuit diagram Table 26,27 and 28.
In principle, the layout of the electric shift control is designed as follows:
From the supply system, the current supply is led through a separation point or distributing box in most cases fitted under the dashbord - to the Controller, and from there to the control unit on the transmission.

The Controller is designed as central selector switch for LH- or RH-Installation.
Reversing is only possible in the 1st Speed.
We recommend to reduce the engine speed at reversing operations.
The Controller is equipped with different ancillaries as Change-back lock, Control of the Converter lockup clutch (WK) and of the Retarder etc.

$1=$ Controller
$2=$ Cable-connection plug
Controller - Transmission - Supply system
$3=$ Hydromedia Reversing transmission
$4=$ elektro hydraulic shift control

EST-25-Family package
(Example Transmission WG-200, Maximum System*)


1 Supply-system connection
2 Display
3 EST-25 Transmission control
4 Cable to the Speed-Solenoid valves
5 Cable to the WK (Converter lockup clutch)
6 Transmission of the WG-Series
7 Cable to the Inductive transmitter - Output
8 Cable to the Inductive transmitter - Turbine
9 Idle switch
10 Load cell
11 Kick-Down switch
12 Controller

The different ZF-Control units can be tested with the Testers PR-47 and PR-78 A + B (see Table $35,36,37,38,39,40,41$ and 42 ) in connection with the special equipment of the actually installed Automatic shift control.
The relative Operating Instructions are delivered together with the corresponding Tester.
These Testing Instructions can be requested from the Service Dept. of the ZF-Passau against a fee of DM 15,-

### 1.5 Retarder:

see Table 3, 13A, 13B, and 13C.
The hydrodynamic retarder (hydraulic retarder), installed upon request, is arranged between engine end torque converter, thus achieving a good braking action in all speeds.
The retarder is a wear-free, gear-dependently acting hydrodynamic brake.
Therefore, the application of the retarder is especially recommended at longer downhill drivings or to brake down from high driving speeds, because herewith the service brake is saved, and in an emergency, the full braking effect (no fading) of the service brake is at disposal.
In this case, it is the question of a so-called primary retarder, i.e. it is frictionally connected with the engine.
Therefore, higher braking torques can be achieved in the lower gear than in the relative higher gears.
Because of this fact, it is necessary that the installed converter lockup clutch must be closed by a control unit, delivered by $\mathbf{Z F}$, so that the full retarder moment can become effective.

### 1.5.1 Layout and Function of the Retarder:

The retarder is composed in its functional parts (see Draft) of the rotor, the stator and the controlling device. The rotor (the rotating part of the retarder) is driven by the overruning vehicle through axle, universal shaft, transmission and shift clutch.
The divided stator is composed of the real stator, which is rigidly connected with the transmission case, and the stator ring. This stator ring is a barrier screen, located between rotor and stationary stator, serving for the reduction of the idle losses.
The controlling device is a valve block, governed by means of air pressure.
In shut-off condition, i.e. without oil filling, the stator ring is hold by elastic force in a position, by which the short-circuit channels between rotor blading and stator blading are created.
In retarder operation, i.e. with oil filling, the stator ring is displaced through the circulating oil, by overcoming the elastic forces, up to a stop.
In this position, it is forming together with the stationary part of the stator a closed blade grid, "contra-rotating" to the rotor.
The braking procedure itself is started through a hand-operated or foot-operated control valve. With this valve, it is possible to adjust a determined air pressure, and with it the desired braking power infinitely variable. In this situation, the possible air-pressure value is lying between 3,5 and 8 bar, because the existing pressurization of the cooling circuit is overcome only from about 3 bar on.
The retarder control valve, which is pressurized by this air pressure, releases in this way the corresponding oil stream into the retarder circuit
Because of the rotation and the blade shape of the rotor, the oil is inversed into the stator, and comes from there back again to the rotor in order to start the circuit anew (see Draft).

Due to the respective inversions, the rotary motion of the rotor and with it the speed of the vehicle will be decelerated.
This energy, transformed in this way from driving speed into pressure, is finally converted by fluid friction into thermal energy and then delivered through a heat exchanger to the cooling water.

If during the retarder operation the shift chutch will be applied, a $3 / 2$-Directional control valve blocks the air-pressure supply to the control valve. In this way, the retarder is shut off and with it, the shift clutch will be saved.


### 1.6 Interaxle differential (lenght-compensating differential):

- see Table 5.


### 1.6.1 Description:

The differential, installed in the final drive, has the task to transmit different torques upon the vehicle axles and to act as intermediate compensation in the driving direction (longitudinal direction of the vehicle).
Due to the existing differential lock, both functions can be put out of service, and a rigid connection between the axles can be realized.

### 1.6.2 Function:

The planetary-gear drive (differential) is designed in such a way that the outboard internal gear is driven.
The drive is realized through the planetary carrier (long lever arm) and through the inboard sun gear (short lever arm).
Since different diameters (lever arm) of planetary carrier (r 1 ) and sun gear (r 2 ) become effective, a torque distribution of $2: 1$ is realized.
Because of the special arrangement of the planetary carriers (Ravignaux-System), the necessary reversion within the planetary-gear drive is realized so that both outputs (planetary carrier and sun gear) are running in the same sense of rotation.

If one axle of the vehicle is slipping, a rigid connection between the vehicle axles can be created through the differential lock.
In this condition, a torque output of $1: 1$ is realized.


### 1.7 Crawler speed: <br> - see Table 4-

For the use in mobile cranes, the converter transmission can be additionally equipped with a crawler-speed stage. In this way, the torque on the transmission output and with it the traction force of the vehicle is increased for $50 \%$, without an additional speed-increasing gear.
The Crawler speed should be used only for the speeds 1 and 2.
An electric control unit protects the function of the Crawler stage.
This unit, for example, takes care that the engagement and disengagement is carried out only at driving speeds below $1 \mathrm{~km} / \mathrm{h}$.

To save the driving axles, it is possible, at engaged Crawler, to drive in the 1st speed only with engaged front axie.


1 Engagement - Disengagement Front axle
2 Planetary speed reduction ( $i=1,5$ )
3 Engagement - Disengagement Crawler speed

### 1.8 Power take-offs:

For the drive of outboard oil pressure pumps, up to 2 engine-dependent power take-offs can be planned, upon request with mechanical and pneumatic disengagement.

The mounting of the pumps with SAE B and C Connections is realized on the output side of the transmission.

### 1.9 Transmission output and Complementary units:

The output is realized through an integrated transfer case. According to the application, one axle can be disengaged. The control can be mechanical or pneumatic.

### 1.10 Emergency steering pump:

The attachment of an emergency steering pump is possible ( $16 \mathrm{~cm}^{3} / \mathrm{rev}$. or $32 \mathrm{~cm}^{3} / \mathrm{rev}$.).
Instead of an emergency steering pump, a secondary pump can be also mounted.
The secondary pump will be output-side driven, and has therefore always a constant delivery direction. At the towing of the vehicle, it supplies the transmission automatically with the necessary quantity of lubricating oil.

### 1.11 Speedometer drive

## Electric Version

The signal necessary for the speedometer evaluation, is inductively picked-up on the output side.

## Mechanical Version

The connection of the speedometer terminal is located on the output gear. The sense of rotation is reversed if the driving direction is changed.

## II. INSTALLATION INSTRUCTIONS

1. Preferred is an elastic 3-(4)-point transmission support.

Tensions in the vehicle frame should not be transmitted upon the transmission case.
2. At a direct mounting on the engine, the end play between converter pin and engine crank shaft, prescribed in the installation drawing, has to be met.
We want to point out that the converter is supporting itself axially upon the crankshaft with 2850 N , at a cold start for a short time up to 5100 N .
3. A torsional vibration damper is necessary if the transmission is separately installed from the engine.
4. Max. three-dimensional working angle of the drive-universal shaft Max. three-dimensional working angle of the output-universal shaft $10^{\circ}$ Shift angles are not permitted!
5. The universal shafts have to be equipped with length compensation, lubrication and shield tube. At movable axles and on the buckling joint, the universal shafts must be equipped in addition with a profile coating and ventilation.
6. The installation position of the joint yokes must be conform with the Catalogue of the Vehicle Manufacturer.
7. Connect the starter interiock.
8. The pressure cut-off of the transmission must become effective only at $30 \%$ of the max. brake pressure (air or oil).
9. Cable hamess installation with sufficient bending radii and abrasion-proof. The plug-and-socket connections should be moisture-proof.
10. Controllable power take-off.

The Gearshift lever must not be under preload in engaged or disengaged condition, the stop must be in the operating mechanism or in the cylinder.
11. At a separate filter installation, the ZF-Filter head has to be used.

As to the filter area or the grade of filtration, the ZF-Recommendation must be observed. The nominal filter flow rate should be about $150 \mathrm{dm}^{3}$.
The line guidance to the external filter must be on the same level or lower than the line connection on the transmission case.
12. If during the use of the vehicle inclined positions greater than $30^{\circ}$ in the longitudinal axis and $15^{\circ}$ in the transverse axis are required, please ask for Information.
13. The required cooling capacity is based on a normal outside temperature of max. $30^{\circ} \mathrm{C}$. The relation of the cooling capacity to the max. converter power (mostly about $80-100 \%$ of the nominal engine power) should be at the minimum:

Caterpillars, Dozers, Wheel loaders, Scrapers
Graders, RT-Cranes, Dumpers $=35-40 \%$
Lift trucks, Road vehicles, Trucks
Vehicles with WK in all speeds $=30-35 \%$
14. At the cooler calculation, corresponding supply-air temperature values must serve as the base for the operation in hot countries.
15. The transmission cooler should be located on the cold-water side of the cooling circuit. If a retarder is used, the total cooling-water quantity must flow through the transmission cooler also at closed thermostatic valve.
16. Cleaning of the lines, filters and cooling system from original dirt (also after a transmission replacement).
17. Internal width of the coooler resp. filter lines and screw connections is at the minimum 32 mm . For line lengths $>1 \mathrm{~m}$, the nominal width must be $+20 \%$.
18. Easy accessibility to the oil filter, oil dipstick and oil filler neck.
19. Monitoring of the control pressure (Measuring point 65) and of the oil temperature (Measuring point 63) in the driver's cab.
20. Use oils according to the ZF-List of lubricants (Oil-level check, see Operating Instruc-) tions).
21. It must be possible, to remove the transmission control unit without removal of the transmission.
22. At welding operations on the vehicle, separate electric or electronic parts from the supply system.
23. Separate ground connections for transmission, control unit and battery must be planned.
24. Pay attention to the specific ZF-Plug assignment, because only then, the use of ZF-Testers will be possible.
25. Connect backup lamps, resp.alarm only through relay.

## NOTE:

The correct installation is the Vehicle Manufacturer's duty!

## III. OPERATION

### 3.1 Driving preparations and Maintenance:

Before the transmission is put into service, take care that the prescribed oil quality is filled in with the correct capacity. At the initial filling of the transmission has to be considered that the oil cooler, the pressure filter as well as the pipes must get filled with oil.
According to these cavities, the cil capacity to be filled in is greater as at the later oil fillings in the course of the usual maintenance program.
Since the converter and also the oil cooler, installed in the vehicle, as well as the pipes can discharge themselves at standstill into the transmission, the Oill-level check must be carried out at engine idle run and operating temperature of the transmission.

## Permitted Transmission oils, see ZF-List of lubricants TE-ML 03, see Table 43!

At the oil-level check, the prescribed Safety Regulations according to $\$ 6$ of the Rules for Accident prevention of power plants in Germany, and in all other countries the respective Country Rules, have to be absolutely repected.
For example, the vehicle must be secured against rolling away with blocks, articulated vehicles must be additionally secured against inadvertent turning-in.

The test has to be carried out at NEUTRAL POSITION of the Controller.
The oil level must be at $80^{\circ} \mathrm{C}$ on the upper mark, up to $40^{\circ} \mathrm{C}$ on the lower mark of the oil dipstick. At the oil-level check, the oil dipstick has to be inserted into the oil-level pipe until contact is obtained. At the oil drain must be considered that only the quantities of oil within the transmission and the upper part of the converter can be drained on the oil drain plug.

At the cleaning of the pressure filter in the main oil stream, pay attention that no dirt or oil sludge can enter into the circuit. Besides, a cover plate has to be planned to protect the output shaft against oil wetting
At the insertion of the filter, every kind of force has to be avoided.

### 3.2 Driving and Shifting:

The starting of the engine has to be always carried out in the NEUTRAL POSITION of the Controller.
For safety reasons, it is recommended to brake the vehicle always securely in position with the parking brake prior to start the engine.

After the starting of the engine and the preselection of the driving direction and the gear, the vehicle can be put into motion by acceleration.
At the drive off, the converter takes over the function of a main clutch.
On a level road, it is possible to drive off also in higher gears.
The acceleration will be only lower.
If the vehicle is stopped and standing with running engine and engaged transmission, the engine cannot be stalled. On a level and horizontal roadway it is possible that the vehicle begins to creep, because the engine is creating through the converter a slight drag torque.
It is advisable to brake the vehicle at every stop securely in position with the parking brake.
At longer standstills, the Controller must be shifted to the NEUTRAL POSITION:

At the drive off, the parking brake must be released. We know by experience that you do not immediately note, in case of a converter transmission, to have forgotten this quite normal operating procedure, because a converter, due to ist high ratio, can easily overcome the bracking torque of the parking brake.
Temperature increases in the converter oil as well as overheated brakes would be the consequences found out later.

We want to point out that the engine speed, due to the braking effect of the converter, is jumping up if speeds are shifted down, especially if one speed is skipped, and can reach in this condition (especially at converters with converter lockup clutch) speeds which are dangerous to the engine. Shifting should be carried out only if the top speed of the lower gear is reached.
If necessary, the vehicle must be braked down to this speed with the foot brake.
This must be also considered at NEUTRAL POSITION of the Controller to avoid a dangerous speed of the manual transmission.

The change-over from NEUTRAL to Reverse, Reverse to Forward or vice versa must be carried out only at vehicle standstill, applied hand brake and at idling speed!

### 3.3 Transmission control with electronic automatic unit:

Exact Informations about the design of the Controllers as well as about the engaged speeds in the single driving ranges have to be taken from the Operating Instructions belonging to the vehicle.

A manual intervention into the automatic shifting procedure (shift through of the driving ranges) is not practical.

In extreme driving situations, e.g. drive off on upgrades, a lower driving range can be preselected and at standstill or after the rolling away of the vehicle immediately shifted in higher driving ranges. This causes the utilization of the lower starting gear without afterward blocking the upshifting in higher gears. If necessary, all speeds of the transmission can be automatically shifted through with this measure (Details, see Operating Instructions).

On certain upgrades, up- and downshiftings may occur in short intervals, namely then if the traction force in the fast gear is too low, however in the slow gear too great for the upgrade.
This shift swinging can be prevented by accelerator throttling or by selection of a lower driving range.

At downhill driving, an unwanted upshifting will be prevented by selection of a lower driving range.
If for the reason of acceleration or at uphill driving the next lower gear shall be engaged, this can be realized by the kick-down position of the accelerator pedal.
This pedal position is causing the earliest possible downshifting in the lower gear.

### 3.4 Control of the transmission output and the various Complementary units:

## Interaxle differential:

The engagement and disengagement of the differential lock must be carried out at the standing vehicle only.

## Axle declutch:

The engagement and disengagement of the front-axle drive must be carried out at the standing vehicle only.
Axle drive APG-8 HK:
The engagement and disengagement of the front-axle drive must be carried out at the standing vehicle only.

## Axle drive DK-7:

Engage front-axle drive
The engagement and disengagement of the 1st and 2nd axle must be carried out at the standing vehicle only.

- Engage axde differential.

If the engagement of the drive of the 1st and 2nd axle is not sufficient to prevent a wheel spinnning, the axle differential can be engaged.

ATTENTION: The engagement must be carried out only at the standing vehicle and the NEUTRAL Position of the gear-selector switch!

- Disengage axle differential:

ATTENTION: As soon as the axle-differential locks as well as the drive of the front axle(s) are no more absolutely necessary for further driving,(solid, antiskid ground), these devices must be immediately disengaged again! At the driving on solid, antiskid ground with engaged front-axle drive or engaged axle-differential locks, considerable defects can occur on the drive axies as well as in the entire drive train.

## Crawler speed:

At the engagement and disengagement of the crawler speed, the following conditions must be considered:

- Version without Front-axle engagement

Engagement and Disengagement

- Neutral
- $<1 \mathrm{~km} / \mathrm{h}$
- Version with Front-axie dechutch

Engagement

- Front-axle drive must be engaged
- Neutral
- $<1 \mathrm{~km} / \mathrm{h}$

Disengagement Crawler speed

- Neutral
- $<1 \mathrm{~km} / \mathrm{h}$

Disengagement Front-axle drive

- Crawler speed out

Since due to the converter there is no rigid connection existing from engine to axle, we recommend to secure the vehicle on upgrades, resp. downgrades against inadvertent rolling away not only by application of the parking brake but additionally by a brake block on the wheel, if the driver has the intention to leave thevehicle.

### 3.6 Towing:

The towing speed at Transmission Versions without secondary pump must be maximally $10 \mathrm{~km} / \mathrm{h}$, and the towing distance not longer than 10 km .
At Transmission Versions with secondary pump, a towing at max. $30 \mathrm{~km} / \mathrm{h}$ up to a distance of 50 km is possible.

## These Instructions must be absolutely respected because otherwise the Transmission becomes defective because of insufficient oil supply!

In case of a longer distance, the defective vehicle must be transported.

### 3.7 Oil temperature:

The oil temperature of the transmission must be controlled by a temperature sensor.
A max. temperature of $120^{\circ} \mathrm{C}$ on the converter exit must not be exceeded.
At a trouble-free unit and an adequate driving style, a higher temperature will not appear.If the temperature is increasing above $120^{\circ} \mathrm{C}$, the vehicle has to be stopped and must be controlled for external oil loss, whilst the engine should be running with a speed of $1200-1500 \mathrm{~min}^{-1}$ at the NEUTRAL POSITION of the transmission.
Now, the temperature must drop quickly (in about 2-3 minutes) to normal values.
If this is not the case, a trouble is existing which must be eliminated prior to continue working.

## Control pressure:

For the supervision of the control pressure, a manometer or a pressure monitor has to be installed. If this pressure is dropping at engaged clutch below the prescribed minimum pressure (at the shifting itself, the pressure will drop for a short time), the reason for the pressure drop must be eliminated. The consequences of a too low control pressure are damages of the clutches which are constantly slipping because of a too low contact force, and are therefore damaged by overheating. On the Schedule of measuring points and the oil circulation diagrams, the mearuring points of the single speeds, the converter pressures and the lubbricating pressure can be seen (see Table 16, $17,18,19,20,21,22,23)$.

## IV. MAINTENANCE

### 4.1 Transmission 6 WG-200:

### 4.1.1 Ol quality:

Permitted for the Powershift transmissions WG-200 are oils according to the ZF-List of lubricants TE-ML 03.
This List of lubricants can be requested from all ZF-Service Statisons.

### 4.1.2 Oil-level check:

Oil-level check (weekly)

- At horizontally standing vehicle
- Transmission in Neutral position ${ }_{4} \mathrm{~N}^{-}$
- Operating temperature of the transmission
- at engine idle run, about $1000 \mathrm{~min}^{4}$
- loosen oil dipstick by anticlockwise rotation, remove and clean it
- insert oil dipstick into the oil level pipe until contact is obtained and take it out
(at least 2x)
- Oil level at $80^{\circ} \mathrm{C}$ on the upper mark ${ }_{\text {HOT }}{ }^{*}$
- Oil level at $40^{\circ} \mathrm{C}$ on the lower mark ${ }_{\mu} C O L D^{4}$


## ATTENTION:

If the oil level has dropped below the Min-Zone, in relation to its condition, it is abolutely necessary to top up oil according to the ZF-List of lubricants, until the oil level is lying between the corresponding temperature marking ~ see Draft -

## NOTE:

According to the Transmission Version, the oil dipstick and the oil filler pipe can have different lengths and shapes. Besides, the fitting on the transmission can be optionally realized on the converter side or on the output side. (The llustration shows the fitting of the oil dipstick on the output side)!

- Insert the oil dipstick again and tighten it by clockwise rotation.



## LEGEND:

1 = OIL FILLER PIPE WTTH OIL DIPSTICK
$2=$ OLLDRAIN PLUG M22 $2 \times 1.5$
-27.
$3=$ OLL DRAIN PLUG M38×1S

### 4.1.3 Oil change and oil filling capacity:

## First oil change after 100 operating hours in action. <br> Every further oil change after 1000 operating hours in action, however at least once a year!

The oil change must be carried out as follows:

- At operating temperature of the transmission, horizontally standing vehicle, open the oil drain plug along with sealing ring and drain the used oil.


## NOTE:

Only the quantities of oil in the transmission and the upper part of the converter can be drained.

- Clean the oil drain plug and the sealing surface on the housing, install it again along with new O-Ring.
- Fill in oil according to the ZF-List of lubricants.
(about 28 liters, sump capacity, external oil quantities, e.g. in the cooler, in the lines etc. are dependent on the vehicle).
The indicated value is a guide value.


## Please pay attention for absolute cleannes of oil and filter!

Binding is at any rate the marking on the oil dipstick!

- start the engine, idle speed
- Transmission in Neutral position ${ }_{n} \mathrm{~N}^{4}$
- top up oil up to the upper marking of the Zone ${ }_{n}$ COLD ${ }^{\prime \prime}$
- brake the vehicle securely in position
- shift all selector switch positions through
- check oil level again, and top up oil again if necessary

Oil level at $80^{\circ} \mathrm{C}$ on the upper marking $\mathrm{HOT}^{\prime \prime}$
Oil level at $40^{\circ} \mathrm{C}$ on the lower marking ${ }^{\circ} \mathrm{COLD}{ }^{4}$, on the oil dipstick.

### 4.1.4 Filter replacement:

The ZF-Fine filter (pressure filter) has to be replaced at every oil change.
Handle the filter with care at the installation, the transportation and the storage!

## Damaged filters must no more be installed!

The installation of the filter must be carried out as follows:

- cover the gasket with a small amount of oil
- Screw the filter in until contact with the sealing surface is obtained and tighten it subsequently by hand $1 / 3$ to $1 / 2$ turn.
- fill in oil
- start the engine
- carry out the transmission oil-level check at the idling engine (about $1000 \mathrm{~min}^{-1}$ ) and at operating temperature of the transmission.:
Oil level at $80^{\circ} \mathrm{C}$ on the upper marking ${ }_{\text {HOT }}{ }^{\boldsymbol{*}}$
Oil level at $40^{\circ} \mathrm{C}$ on the lower marking ${ }^{\circ}$ COLD", on the oil dipstick.
- check tightness, if necessary tighten by hand again.


### 4.2 Axle drive DK7:

### 4.2.1 Oil quality:

Permitted are oils according to the ZF-List of lubricants TE-ML 05, see Table 44.
This List of lubricants can be requested from all ZF-Service Stations.

LEGEND:
$\mathrm{A}=\mathrm{Oil}$ filler serew M24×1,5
$\mathrm{B}=$ Oil filler screw M24×1,5
$\mathrm{C}=$ Oil filler screw and oil-level check M36x1,5
$1=$ Oil drain plag M24×1,5
$2=$ Oil drain plug M24×1.5
$3=$ Oil drain plug M $24 \times 1,5$

### 4.2.2 Oil-level check (weekdy):

- At horizontally standing vehicle, open the screw plug (C) on the transmission. the oil level must reach up to the lower edge of the check hole (C). If necessary, top up oil according to the ZF-List of lubricants TE-ML 05

ATTENTION: The operating oil level is located at (C).
If the oil level has fallen, it is absolutely necessary to top up oil,according to the ZF-List of lubricants, up to the overflow on the oil-level check hole ( C ).
If no oil is replenished, there is a danger of a transmission defect!

- Clean the sealing surfaces on the housing and on the screw plug.
- Screw the screw plug, equipped with new O-Ring, again into the oil-level check hole (C) and tighten.


### 4.2.3 Oil change:

## ATTENTION: At all kinds of work, employ utmost cleanliness so that no dirt can pe-

 netrate into the transmission interior!- At horizontally standing vehicle, open the screw plugs ( $1,2,3$ ) and drain the used oil. Clean screw plugs $(1,2,3)$ and sealing surfaces on the housing, install screw plugs equipped with new $O$-Rings again and tighten.
- Open screw plugs (A, B, C) and clean sealing surfaces.
- Fill in oil according to the ZF-List of lubricants TE-ML. 05 at ( $A$ and $B$ ) until it begins to overflow on the oil-level check hole (C)


## At the filling, the following order must be observed:

1. Oil filling (A)
2. Oil filling (B)

### 4.3 Axle drive APG-8/HK:

### 4.3.1 Oil quality:

Permitted are oils according to the ZF-List of lubricants TE-ML 05 This List of lubricants can be requested from all ZF-Service Stations.


## LEGEND

$1=$ Oil filler plug and oil-level check M42x1,5
$2=$ Oil drain plug M $24 \times 1,5$

## 4,3.2 Oil-level check (weekly)

- At horizontally standing vehicle, open the screw plug (1) on the transmission, the oil level must reach up to the lower edge of the check hole.
If necessary, top up oil according to the ZF-List of lubricants TE-ML 05,
- Clean sealing surfaces on the housing and on the screw plug.
- Install screw plug, equipped with new 0-Ring again into the oil-level check hole and tighten.


### 4.3.3 Oil change on the Axle drive:

ATTENTION: At all kinds of work, employ utmost cleanliness, so that no dirt can penetrate into the transmission interior!

- At horizontally standing vehicle, open the screw plug (2) and drain the used oil.
- Clean screw plug and sealing surfaces on the housing, install serew plug equipped with new O-Ring again and tighten.
- Open screw plug (1) and clean sealing surfaces.
- Fill in oil according to the ZF-List of lubricants TE-ML. 05, until it begins to overflow on the oil-level check hole.
- Install screw plug (1) equipped with new O-Ring again and tighten.


## LAYOUT 6 WG-200 WITH AXLE DECLUTCH TABLE - 1

$1=$ ENGINE CONNECTION
$2=\mathrm{WK}$
3 = CONVERTER
4 = CLUTCH KV
5 = BLEEDER
6 = CLUTCH K1
7 = 2nd POWER TAKE-OFF
8 = CONVERTER CHARGE AND CONTROL PRESSURE PUMP
$9=1$ st POWER TAKE-OFF
$10=$ CLUTCH K2
$11=$ CLUTCH K3
$12=$ SPEEDOMETER DRIVE
$13=$ OUTPUT - REAR AXLE
14 = OIL SUMP
$15=$ AXLE DECLUTCH - FRONT AXLE
$16=$ LAYSHAFT
$17=$ CLUTCH K4
$18=$ CLUTCH KR


LAYOUT 6 WG-200 WITH INTERAXLE DIFFERENTIAL TABLE - 2

1 = 2nd POWER TAKE-OFF
$2=$ CLUTCH K1
3 = CONVERTER CHARGE AND CONTROL PRESSURE PUMP
$4=1$ st POWER TAKE-OFF
5 = CLUTCH K2
6 = CLUTCH K3
7 = INTERAXLE DIFFERENTIAL
8 = BRAKE DISK
9 = OUTPUT - REAR AXLE
$10=$ OIL SUMP
$11=$ ENGAGEMENT AND DISENGAGEMENT DIFFERENTIAL LOCK
$12=$ EMERGENCY STEERING PUMP
$13=$ LAYSHAFT
$14=$ CLUTCH K4
$15=$ CLUTCH KR
$16=$ ENGINE CONNECTION
$17=$ CONVERTER
$18=\mathrm{WK}$
$19=$ CLUTCH KV


## RETARDER 6 WG-200

 TABLE - 3```
1 = RETARDER
2 = CONVERTER WITH WK
3 = DRIVE SHAFT
4 = DRIVE FLANGE
5 = THRUST RING
6 = ROTOR
7 = STATOR RING
8 = STATOR
9 = COVER
10= WK
11= CONVERTER
12 = CONVERTER HOUSING COVER
```



## CRAWLER SPEED 6 WG-200

TABLE - 4

1 = OUTPUT - FRONT AXLE
2 = DRIVER
3 = TRANSMISSION CASE 6-WG 200
$4=$ ENGAGEMENT AND DISENGAGEMENT FRONT AXLE
5 = INTERNAL GEAR/INTERNAL-GEAR CARRIER
6 = PLANETARY SHAFT/PLANETARY GEAR
7 = PLANETARY CARRIER
$8=$ HOLLOW SHAFT
9 = ENGAGEMENT AND DISENGAGEMENT CRAWLER SPEED
$10=$ CLUTCH FLANGE
$11=$ OUTPUT - REAR AXLE
$12=$ SPEEDOMETER DRIVE
13 = HYDRAULIC ENGAGEMENT AND DISENGAGEMENT CRAWLER SPEED
14 = SUN GEAR
$15=$ HYDRAULIC ENGAGEMENT AND DISENGAGEMENT FRONT AXLE


INTERAXLE DIFFERENTIAL 6 WG-200
TABLE - 5
$1=$ OUTPUT - FRONT AXLE
2 = ENGAGEMENT AND DISENGAGEMENT DIFFERENTIAL LOCK FRONT AXLE
3 = HOLLOW SHAFT
4 = PLANETARY CARRIER
5 = PLANETARY SHAFT/PLANETARY GEAR (RAVIGNAUX-SYSTEM OUTER HOLE CIRCLE)
$6=$ SUN GEAR
7 = OUTPUT - REAR AXLE
8 = ATTACHMENT POSSIBILITY SPEEDOMETER
9 = SUPPORTING PLATE/DRIVE GEAR/SUPPORTING TUBE
$10=$ PLANETARY SHAFT/PLANETARY GEAR (RAVIGNAUX-SYSTEM INNER HOLE CIRCLE)
$11=$ MECHANICAL ENGAGEMENT AND DISENGAGEMENT OF THE DIFFERENTIAL LOCK


## AXLE DRIVE DK-7

 FLANGE-MOUNTED ON 6 WG-200 TABLE - 6$1=$ HYDRAULIC ENGAGEMENT AND DISENGAGEMENT OF THE FRONT AXLE
$2=$ SWITCH DISPLAY DISENGAGEMENT FRONT AXLE
3 = ENGAGEMENT AND DISENGAGEMENT FRONT AXLE
$4=$ HOLLOW SHAFT
$5=$ ENGAGEMENT AND DISENGAGEMENT AXLE DIFFERENTIAL
6 = OUTPUT TO THE CENTER AXLE - RIGHT
7 = DRIVE SHAFT
$8=$ HYDRAULIC ENGAGEMENT AND DISENGAGEMENT OF THE AXLE DIFFERENTIAL
$9=$ SWITCH DISPLAY DIFFERENTIAL LOCK
$10=$ ADAPTER TRANSMISSION/AXLE DRIVE DK-7
$11=$ OUTPUT SHAFT
$12=$ TRANSMISSION CASE 6 WG-200
$13=$ ENGAGEMENT AND DISENGAGEMENT FRONT AND CENTER AXLE
$14=$ CRAWLER SPEED
$15=$ OUTPUT SHAFT TO THE REAR AXLE
$16=$ HYDRAULIC ENGAGEMENT AND DISENGAGEMENT OF THE FRONT AND CENTER AXLE
$17=$ DIFFERENTIAL - CENTER AXLE
$18=$ OUTPUT TO THE CENTER AXLE - LEFT
$19=$ BEVEL-GEAR SET - DRIVE CENTER AXLE
$20=$ SPUR-GEAR DRIVE - CENTER AXLE
$21=$ OUTPUT TO THE FRONT AXLE


## AXLE DRIVE APG-8/HK FLANGE-MOUNTED ON 6 WG-200 TABLE - 7

1 = OUTPUT FRONT AXLE - RIGHT
$2=$ BEVEL-GEAR SET
3 - ENGAGEMENT AND DISENGAGEMENT OF THE INTERAXLE DIFFERENTIAL
$4=$ INTERAXLE DIFFERENTIAL
$5=$ OUTPUT REAR AXLE
$6=$ TRANSMISSION CASE 6 WG-200
7 - VERSION WITHOUT MULTI-DISK SELF-LOCKING DIFFERENTIAL
$8=$ OUTPUT FRONT AXLE - LEFT
$9=$ VERSION WITH MULTI-DISK SELF-LOCKING DIFFERENTIAL


## INSTALLATION VIEW 6WG-200 CENTER DISTANCE 400 MM FRONT VIEW

TABLE - 8A

1 = CONNECTION FROM HEAT EXCHANGER
$2=$ LIFTING LUGS
3 = ENGINE CONNECTION - DIRECT MOUNTING
$4=$ ELECTRIC SHIFT CONTROL
$5=$ TRANSMISSION SUSPENSION POINTS
6 = OIL SUCTION TUBE
7 = OUTPUT - FRONT AXLE
8 = ATTACHMENT POSSIBILITY FOR OIL-LEVEL PIPE - WITH OIL DIPSTICK
9 = MARKING OIL LEVEL
$10=$ INDUCTIVE TRANSMITTER FOR TURBINE SPEED


## INSTALLATION VIEW 6WG-200 CENTER DISTANCE 400 MM SIDE VIEW

TABLE - 8B

1 = ENGINE CONNECTION DIRECT MOUNTING
2 = BLEEDER
3 = PLUG CONNECTION - CANNON
4 = BOX FILTER
5 WK HOSE LINE - CONNECTION ON
WK-VALVE
6 = ELECTRIC SHIFT CONTROL
7 = SPEEDOMETER - MECHANICAL
8 = OUTPUT REAR AXLE
9 = OIL SUCTION PIPE
$10=$ MODEL IDENTIFICATION PLATE
$11=$ ATTACHMENT POSSIBILITY FOR OIL FILLER PIPE WITH OIL DIPSTICK
$12=$ OUTPUT - FRONT AXLE
$13=$ SPEEDOMETER - ELECTRONIC
$14=$ CONNECTION TO HEAT EXCHANGER
$15=$ TEMPERATURE SWITCH MEASURING POINT BEHIND THE CONVERTER
16 = WK HOSE LINE - CONNECTION ON OIL SUPPLY FLANGE


## INSTALLATION VIEW 6 WG-200 CENTER DISTANCE 400 MM REAR VIEW <br> TABLE - 8C

$1=$ WK HOSE LINE - CONNECTION ON WK-VALVE
2 = BOX FILTER
$3=1$ st POWER TAKE-OFF
4 = HOSE LINE KV
(CONNECTION ON AXLE AND CHANNEL PLATE)
5 = INDUCTIVE TRANSMITTER FOR TURBINE SPEED
6 = HOSE LINE K1
(CONNECTION ON AXLE AND CHANNEL PLATE)
7 = HOSE LINE KR
(CONNECTION ON AXLE AND CHANNEL PLATE)
8 = HOSE LINE K2
(CONNECTION ON AXLE AND CHANNEL PLATE)
9 = OIL DRAIN PLUG
$10=$ OUTPUT - REAR AXLE
$11=$ HOSE LINE K3
(CONNECTION ON AXLE AND CHANNEL PLATE)
$12=$ HOSE LINE K4
(CONNECTION ON AXLE AND CHANNEL PLATE)


INSTALLATION VIEW 6 WG-200 CENTER DISTANCE 500 MM FRONT VIEW
TAFEL - 9A

1 = BLEEDER
2 = LIFTING LUGS
3 = ENGINE CONNECTION DIRECT MOUNTING
4 = ELECTRIC SHIFT CONTROL
5 = TEMPERATURE SWITCH (MEASURING POINT BEHIND THE CONVERTER
$6=$ TRANSMISSION SUSPENSION POINTS
7 = OIL SUCTION PIPE
8 = AXLE DECLUTCH - FRONT AXLE
9 = OIL DRAIN PLUG
$10=$ MARKING OIL LEVEL
$11=$ OUTPUT - FRONT AXLE
$12=$ CONNECTION FROM HEAT EXCHANGER
$13=$ INDUCTIVE TRANSMITTER TURBINE SPEED


INSTALLATION VIEW 6 WG-200
CENTER DISTANCE 500 MM SIDE VIEW
TABLE - 9B

1 = ENGINE CONNECTION - DIRECT MOUNTING
2 = BLEEDER
3 = PLUG CONNECTION - CANNON
4 = WK HOSE LINE - CONNECTION ON wK-Valve
5 = BOX FILTER
6 = ELECTRIC SHIFT CONTROL
7 = EMERGENCY STEERING PUMP
$8=$ SPEEDOMETER - MECHANICAL
9 = OUTPUT - REAR AXLE
$10=$ SPEEDOMETER - ELECTRONIC
$11=$ OIL DRAIN PLUG
$12=$ OIL SUCTION PIPE
$13=$ MODEL IDENTIFICATION PLATE
$14=$ ATTACHMENT POSSIBILITY FOR OIL FILLER PIPE WITH OIL DIPSTICK
$15=$ AXLE DECLUTCH - FRONT AXLE
16= OUTPUT FRONT AXLE
$17=$ CONNECTION TO HEAT EXCHANGER
$18=$ TEMPERATURE SWITCH MEASURING POINT BEHIND THE CONVERTER
$19=$ WK HOSE LINE - CONNECTION ON OIL SUPPLY FLANGE


## INSTALLATION VIEW 6 WG-200

 CENTER DISTANCE 500 MM REAR VIEW TABLE - 9C1 = WK HOSE LINE - CONNECTION ON WK-VALVE
2 = BOX FILTER
3 = 1st POWER TAKE-OFF
4 = 2nd POWER TAKE-OFF
5 = HOSE LINE KV
(CONNECTION ON AXLE AND CHANNEL PLATE)
6 = INDUCTIVE TRANSMITTER FOR TURBINE SPEED
7 = HOSE LINE K1
(CONNECTION ON AXLE AND CHANNEL PLATE)
8 = EMERGENCY STEERING PUMP
9 = HOSE LINE K4
(CONNECTION ON AXLE AND CHANNEL PLATE)
$10=$ HOSE LINE K3
(CONNECTION ON AXLE AND CHANNEL PLATE)
$11=$ ATTACHMENT POSSIBILITY FOR OIL-LEVEL PIPE WITH OIL DIPSTICK
$12=$ OUTPUT REAR AXLE
$13=$ INDUCTIVE TRANSMITTER FOR OUTPUT SPEED (SPEEDOMETER)
$14=$ HOSE LINE K2
(CONNECTION ON AXLE AND CHANNEL PLATE)
$15=$ HOSE LINE KR
(CONNECTION ON AXLE AND CHANNEL PLATE)


## INSTALLATION VIEW 6 WG-200 DIRECT MOUNTING CENTER DISTANCE 555 MM

 FRONT VIEW TABLE - 10A1 = ENGINE CONNECTION - DIRECT MOUNTING
2 = ELECTRIC SHIFT CONTROL
$3=$ TRANSMISSION SUSPENSION POINTS
4 = OIL SUCTION PIPE
5 = AXLE DECLUTCH - FRONT AXLE
6 = OUTPUT - FRONT AXLE
7 = OIL DRAIN PLUG
8 = MARKING OIL LEVEL
$9=$ CONNECTION FROM HEAT EXCHANGER
$10=$ INDUCTIVE TRANSMITTER TURBINE SPEED


INSTALLATION VIEW 6 WG-200 DIRECT MOUNTING CENTER DISTANCE 555 MM SIDE VIEW
TABLE - 10B

1 = ENGINE CONNECTION - DIRECT MOUNTING
2 = LIFTING LUGS
3 = BLEEDER
$4=$ PLUG CONNECTION - CANNON
5 = WK HOSE LINE - CONNECTION ON WK-VALVE
6 = ELECTRIC SHIFT CONTROL
7 = EMERGENCY STEERING PUMP
8 = SPEEDOMETER - MECHANICAL
9 = OUTPUT - REAR AXLE
$10=$ MODEL IDENTIFICATION PLATE
11 = SPEEDOMETER - ELECTRONIC
$12=$ OIL SUCTION PIPE
$13=$ AXLE DECLUTCH - FRONT AXLE
14 = OUTPUT - FRONT AXLE
$15=$ CONNECTION TO HEAT EXCHANGER
$16=$ TEMPERATURE SWITCH MEASURING POINT BEHIND THE CONVERTER
$17=$ WK HOSE LINE - CONNECTION ON OIL SUPPLY FLANGE


## CENTER DISTANCE 555 MM REAR VIEW TABLE - 10C

INSTALLATION VIEW 6 WG-200 DIRECT MOUNTING

1 = WK HOSE LINE - CONNECTION ON WK-VALVE
2 = BLEEDER
3 = CONNECTION TO BOX FILTER - SEPARATE INSTALLATION
$4=$ CONNECTION FROM BOX FILTER - SEPARATE INSTALLATION
$5=1$ st POWER TAKE-OFF
6 = 2nd POWER TAKE-OFF
7 = HOSE LINE KV
(CONNECTION ON AXLE AND CHANNEL PLATE)
8 = INDUCTIVE TRANSMITTER FOR TURBINE SPEED
9 = HOSE LINE K1
(CONNECTION ON AXLE AND CHANNEL PLATE)
$10=$ EMERGENCY STEERING PUMP
11 = HOSE LINE K3
(CONNECTION ON AXLE AND CHANNEL PLATE)
$12=$ HOSE LINE K4
(CONNECTION ON AXLE AND CHANNEL PLATE)
$13=$ ATTACHMENT POSSIBILITY FOR OIL-LEVEL PIPE
WITH OIL DIPSTICK
14 = OUTPUT - REAR AXLE
$15=$ SPEEDOMETER DRIVE - MECHANICAL
$16=$ HOSE LINE K2
(CONNECTION ON AXLE AND CHANNEL PLATE)
17 = HOSE LINE KR
(CONNECTION ON AXLE AND CHANNEL PLATE)


INSTALLATION VIEW 6 WG-200 SEPARATE INSTALLATION CENTER DISTANCE 555 MM FRONT VIEW
TABLE-11A

1 = DRIVE FLANGE - DRIVE THROUGH UNIVERSAL SHAFT
2 = ELECTRIC SHIFT CONTROL
3 = TRANSMISSION SUSPENSION POINTS
4 = OIL SUCTION PIPE
$5=$ ENGAGEMENT AND DISENGAGEMENT INTERAXLE DIFFERENTIAL
6 = OUTPUT - FRONT AXLE
7 = OIL DRAIN PLUG
8 = MARKING OIL LEVEL
$9=$ CONNECTION FROM HEAT EXCHANGER
$10=$ INDUCTIVE TRANSMITTER TURBINE SPEED


INSTALLATION VIEW 6 WG-200 SEPARATE INSTALLATION CENTER DISTANCE 555 MM SIDE VIEW
TABLE-11B

1 = WK HOSE LINE - CONNECTION ON OIL SUPPLY FLANGE
2 = LIFTING LUGS
3 = BLEEDER
4 = PLUG CONNECTION - CANNON
5 = WK HOSE LINE - CONNECTION ON WK-VALVE
6 = ELECTRIC SHIFT CONTROL
7 = SPEEDOMETER - MECHANICAL
8 = OUTPUT - REAR AXLE
$9=$ MODEL IDENTIFICATION PLATE
$10=$ SPEEDOMETER - ELECTRONIC
11 = OIL SUCTION PIPE
$12=$ ENGAGEMENT AND DISENGAGEMENT INTERAXLE DIFFERENTIAL
$13=$ OUTPUT - FRONT AXLE
$14=$ CONNECTION TO HEAT EXCHANGER
$15=$ TEMPERATURE SWITCH MEASURING POINT BEHIND THE CONVERTER
$16=$ DRIVE FLANGE - DRIVE THROUGH UNIVERSAL SHAFT


INSTALLATION VIEW 6 WG-200 SEPARATE INSTALLATION CENTER DISTANCE 555 MM REAR VIEW TABLE - 11C

1 = WK HOSE LINE - CONNECTION ON WK-VALVE
2 = BLEEDER
3 = BOX FILTER
$4=1$ st POWER TAKE-OFF
$5=2$ nd POWER TAKE-OFF
6 = HOSE LINE KV
(CONNECTION ON AXLE AND CHANNEL PLATE)
7 = INDUCTIVE TRANSMITTER FOR TURBINE SPEED
8 = HOSE LINE K1
(CONNECTION ON AXLE AND CHANNEL PLATE)
9 = ATTACHMENT POSSIBILITY FOR EMERGENCY STEERING PUMP
$10=$ INDUCTIVE TRANSMITTER FOR OUTPUT SPEED
11 = HOSE LINE K3
(CONNECTION ON AXLE AND CHANNEL PLATE)
$12=$ HOSE LINE K4
(CONNECTION ON AXLE AND CHANNEL PLATE)
$13=$ OIL FILLER PIPE WITH OIL DIPSTICK
14 = OUTPUT - REAR AXLE
$15=$ SPEEDOMETER DRIVE - MECHANICAL
$16=$ HOSE LINE K2
(CONNECTION ON AXLE AND CHANNEL PLATE)
$17=$ HOSE LINE KR
(CONNECTION ON AXLE AND CHANNEL PLATE)


INSTALLATION VIEW 6 WG-200 SHIFT CONTROL LOWER POSITION CENTER DISTANCE 555 MM FRONT VIEW
TABLE - 12A

1 = DRIVE FLANGE - DRIVE THROUGH UNIVERSAL SHAFT
$2=$ ELECTRIC SHIFT CONTROL
3 = TRANSMISSION SUSPENSION POINTS
4 = OUTPUT - FRONT AXLE
5 = OIL SUCTION PIPE
6 = AXLE DECLUTCH - FRONT AXLE
7 = OIL DRAIN PLUG
8 = MARKING OIL LEVEL
$9=$ OIL FILLING PIPE WITH OIL DIPSTICK
$10=$ INDUCTIVE TRANSMITTER TURBINE SPEED


## INSTALLATION VIEW 6 WG-200 SHIFT CONTROL LOWER POSITION

 CENTER DISTANCE 555 MM SIDE VIEWTABLE - 12B

1 = DRIVE FLANGE - DRIVE THROUGH UNIVERSAL SHAFT
2 = HOSE LINE - FROM SHIFT CONTROL TO CONVERTER
3 = HOSE LINE - FROM BOX FILTER TO SHIFT CONTROL
4 = LIFTING LUGS
5 = BLEEDER
6 = BOX FILTER
7 = CONNECTION TO HEAT EXCHANGER
8 = EMERGENCY STEERING PUMP
9 = SPEEDOMETER - MECHANICAL
$10=$ OUTPUT - REAR AXLE
$11=$ MODEL IDENTIFICATION PLATE
$12=$ SPEEDOMETER - ELECTRONIC
$13=$ OIL SUCTION PIPE
14 = AXLE DECLUTCH - FRONT AXLE
$15=$ OUTPUT - FRONT AXLE
$16=$ HOSE LINE - FROM THE SHIFT CONTROL TO THE SUMP
$17=$ HOSE LINE - FROM SHIFT CONTROL HOUSING COVER TO THE SUMP (VENTILATION)


INSTALLATION VIEW 6 WG-200 SHIFT CONTROL LOWER POSITION CENTER DISTANCE 555 MM REAR VIEW
TABLE - 12C

1 = CONNECTION - HOSE LINE FROM BOX FILTER TO SHIFT CONTROL
2 = BLEEDER
3 = BOX FILTER
4 = LIFTING LUGS
5 = 1st POWER TAKE-OFF
6 = 2nd POWER TAKE-OFF
7 = HOSE LINE KV
$8=$ INDUCTIVE TRANSMITTER FOR TURBINE SPEED
9 = HOSE LINE K1
$10=$ EMERGENCY STEERING PUMP
11 = OIL FILLER PIPE WITH OIL DIPSTICK
$12=$ OUTPUT - REAR AXLE
$13=$ CONNECTION OF THE HOSE LINES FROM THE SHIFT CONTROL AND SHIFT CONTROL HOUSING COVER TO SUMP
$14=$ HOSE LINE K4
$15=$ HOSE LINE K3
$16=$ HOSE LINE K2
$17=$ HOSE LINE KR


INSTALLATION VIEW 6 WG-200 WITH RETARDER CENTER DISTANCE 555 MM FRONT VIEW
TABLE-13A

1 = DRIVE FLANGE - DRIVE THROUGH UNIVERSAL SHAFT
$2=$ ELECTRIC SHIFT CONTROL
3 = WK-HOSE LINE - CONNECTION ON WK-VALVE
4 = OIL LINE FROM - IN FRONT OF THE CONVERTER TO THE RETARDER VALVE
5 = RETARDER VALVE
6 = TEMPERATURE SWITCH - MEASURING POINT BEHIND THE CONVERTER
7 = ENGAGEMENT AND DISENGAGEMENT INTERAXLE DIFFERENTIAL
8 = OIL SUCTION PIPE
$9=$ OUTPUT - FRONT AXLE
$10=$ OIL DRAIN PLUG
$11=$ MARKING OIL LEVEL
$12=$ OIL LINE FROM THE HEAT EXCHANGER TO THE LUBRICATION
$13=$ OIL FILLER PIPE WITH OIL DIPSTICK
$14=$ TRANSMISSION SUSPENSION POINTS
$15=$ INDUCTIVE TRANSMITTER TURBINE SPEED


## INSTALLATION VIEW 6 WG-200 WITH RETARDER

 CENTER DISTANCE 555 MM SIDE VIEWTABLE-13B

1 = DRIVE FLANGE - DRIVE THROUGH UNIVERSAL SHAFT
$2=$ STATOR
3 = CONVERTER HOUSING COVER
4 = WK-HOSE LINE - TO OIL SUPPLY FLANGE
$5=$ BLEEDER
$6=$ BOX FILTER
$7=$ ELECTRIC SHIFT CONTROL
8 = WK-HOSE LINE - CONNECTION ON WK-VALVE
$9=$ HOSE LINE FROM HEAT EXCHANGER TO LUBRICATION
$10=$ HOSE LINE - FROM SHIFT CONTROL TO SUMP
$11=$ SPEEDOMETER - MECHANICAL
$12=$ OUTPUT - REAR AXLE WITH BRAKE DISK
$13=$ OIL FILLER AND CHECK PIPE
$14=$ MODEL IDENTIFICATION PLATE
$15=$ SPEEDOMETER - ELECTRONIC
$16=$ OIL SUCTION PIPE
$17=$ ENGAGEMENT AND DISENGAGEMENT INTERAXLE DIFFERENTIAL
18 = OUTPUT - FRONT AXLE
$19=$ HOSE LINE FROM - BEHIND THE CONVERTER -
TO THE RETARDER VALVE
$20=$ RETARDER VALVE
$21=$ HOSE LINE FROM - IN FRONT OF THE CONVERTER TO THE RETARDER VALVE


## CENTER DISTANCE 555 MM REAR VIEW TABLE-13C

INSTALLATION VIEW 6 WG-200 WITH RETARDER

1 = BLEEDER
2 = BOX FILTER
3 = LIFTING LUGS
$4=1$ st POWER TAKE-OFF
5 = HOSE LINE KV
(CONNECTION ON AXLE AND CHANNEL PLATE)
6 = INDUCTIVE TRANSMITTER FOR TURBINE SPEED
7 = HOSE LINE K1
(CONNECTION ON AXLE AND CHANNEL PLATE)
8 = ATTACHMENT POSSIBILITY FOR EMERGENCY STEERING PUMP
9 = INDUCTIVE TRANSMITTER FOR OUTPUT SPEED
$10=$ OIL FILLER PIPE WITH OIL DIPSTICK
11 = OUTPUT - REAR AXLE WITH BRAKE DISK
$12=$ HOSE LINE - LEAK OIL
$13=$ HOSE LINE K4
(CONNECTION ON AXLE AND CHANNEL PLATE)
$14=$ HOSE LINE K3
(CONNECTION ON AXLE AND CHANNEL PLATE)
$15=$ HOSE LINE K2
(CONNECTION ON AXLE AND CHANNEL PLATE)
16 = HOSE LINE KR
(CONNECTION ON AXLE AND CHANNEL PLATE
$17=$ WK HOSE LINE - CONNECTION ON WK-VALVE


## INSTALLATION VIEW 6 WG-200 WITH AXLE DRIVE APG-8/HK

 CENTER DISTANCE 555 MM FRONT VIEWTABLE - 14A

1 = DRIVE FLANGE - DRIVE THROUGH UNIVERSAL SHAFT
2 = ELECTRIC SHIFT CONTROL
3 = WK-HOSE LINE - CONNECTION ON WK-VALVE
4 = OIL SUCTION PIPE
5 = OUTPUT - FRONT AXLE LEFT
6 = OIL FILLER AND OIL-LEVEL CHECK PLUG AXLE DRIVE APG-8/HK
7 = EMERGENCY STEERING PUMP
8 = OUTPUT - FRONT AXLE RIGHT
9 = AXLE DRIVE - APG-8/HK
$10=$ CONNECTION FROM HEAT EXCHANGER
$11=$ OIL FILLER PIPE WITH OIL DIPSTICK FOR THE TRANSMISSION 6 WG-200
$12=$ INDUCTIVE TRANSMITTER TURBINE SPEED


INSTALLATION VIEW 6 WG-200 WITH AXLE DRIVE APG-8/HK CENTER DISTANCE 555 MM SIDE VIEW
TABLE - 14B

1 = DRIVE FLANGE - DRIVE THROUGH UNIVERSAL SHAFT
2 = CONVERTER HOUSING COVER
3 = LIFTING LUGS
4 = BLEEDER
5 = PLUG CONNECTION - CANNON
6 = ELECTRIC SHIFT CONTROL
7 = WK HOSE LINE - CONNECTION ON WK-VALVE
8 = CONNECTION TO THE HEAT EXCHANGER
9 = SPEEDOMETER - MECHANICAL
$10=$ OUTPUT REAR AXLE
$11=$ MODEL IDENTIFICATION PLATE
$12=$ SPEEDOMETER - ELECTRONIC
$13=$ OIL SUCTION PIPE
$14=$ OIL DRAIN PLUG TRANSMISSION
$15=$ ENGAGEMENT/DISENGAGEMENT INTERAXLE DIFF.
$16=$ CONN. SUCTION LINE EMERGENCY STEERING PUMP
17 = OIL DRAIN PLUG - AXLE DRIVE APG-8/HK
$18=$ OIL FILLER AND OIL-LEVEL CHECK PLUG AXLE DRIVE APG-8/HK
$19=$ OUTPUT - FRONT AXLE LEFT
$20=$ AXLE DRIVE APG-8/HK
$21=$ TEMP. SWITCH MEASURING POINTS BEHIND CONVERTER


## INSTALLATION VIEW 6 WG-200 WITH AXLE DRIVE APG-8/HK

 CENTER DISTANCE 555 MM REAR VIEW TABLE - 14C$1=$ CONNECTION TO BOX FILTER - SEPARATE INSTALLTION
2 = BLEEDER
3 = CONNECTION FROM BOX FILTER - SEPARATE INSTALLATION
4 = LIFTING LUGS
5 = 1st POWER TAKE-OFF
6 = HOSE LINE KV
(CONNECTION ON AXLE AND CHANNEL PLATE)
7 = INDUCTIVE TRANSMITTER FOR TURBINE SPEED
8 = HOSE LINE K1
(CONNECTION ON AXLE AND CHANNEL PLATE)
9 = OIL FILLER PIPE WITH OIL DIPSTICK FOR
THE TRANSMISSION 6 WG-200
$10=$ HOSE LINE K3
(CONNECTION ON AXLE AND CHANNEL PLATE)
11 = HOSE LINE K4
(CONNECTION ON AXLE AND CHANNEL PLATE)
$12=$ OUTPUT - FRONT AXLE RIGHT
$13=$ OUTPUT - REAR AXLE
$14=$ SPEEDOMETER - MECHANICAL
$15=$ HOSE LINE K2
(CONNECTION ON AXLE AND CHANNEL PLATE)
$16=$ HOSE LINE KR
(CONNECTION ON AXLE AND CHANNEL PLATE)


17 = WK HOSE LINE - CONNECTION ON WK-VALVE

## INSTALLATION VIEW 6 WG-200 WITH AXLE DRIVE DK-7 AND

 CRAWLER SPEED CENTER DISTANCE 555 MM FRONT VIEWTABLE - 15A

1 = OIL FILLER PIPE WITH OIL DIPSTICK FOR THE TRANSMISSION 6 WG-200
2 = DRIVE FLANGE - DRIVE THROUGH UNIVERSAL SHAFT
3 = BLEEDER
4 = ELECTRIC SHFT CONTROL
5 = CONSOLE
6 = AXLE DRIVE DK-7
7 = OUTPUT - CENTER AXLE RIGHT
8 = OUTPUT - FRONT AXLE
$9=$ INDUCTIVE TRANSMITTER TURBINE SPEED




## SCHEDULE OF MEASURING POINTS 6 WG-200 WITH ALUMINUM DIE-CASTING SHIFT CONTROL; WK-VALVE AND VARIOVALVE

## TABLE - 16

THE MARKED POSITIONS (e.g. 53) CORRESPOND WITH THE POSITIONS ON TABLE 20 !!
THE MEASUREMENTS HAVE TO BE CARRIED OUT AT HOT TRANSMISSION (ABOUT $80^{\circ}-95^{\circ} \mathrm{C}$ ) !

MEASURING POINTS FOR PRESSURE OIL AND TEMPERATURE
51 = IN FRONT OF THE CONVERTER
OPENING PRESSURE 8,5 bar
$53=$ CLUTCH FORWARD
$55=$ CLUTCH REVERSE
$56=$ CLUTCH
$57=$ CLUTCH
$58=$ CLUTCH
$60=$ CLUTCH
$63=$ BEHIND THE CONVERTER AND
OPENING PRESSURE 2,5 bar
$65=$ SYSTEM PRESSURE $16+2$ bar
$66=$ REDUCING VALVE 10 bar
$67=$ WK CONTROL PRESSURE $13 \pm 1$ bar

## MEASURING POINTS FOR DELIVERY RATES

## $15=$ CONNECTION TO HEAT EXCHANGER

$16=$ CONNECTION FROM HEAT EXCHNGER

TRANSMISSION DIAGRAM


[^1]M26x1,5

## NDUCTIVE TRANSMITTER

$1=$ INDUCTIVE TRANSMITTER TURBINE SPEED M18x1,5
$2=$ INDUCTIVE TRANSMITTER OUTPUT SPEED M18x1,5

## SOLENOID VALVES <br> M6 = WK-VALVE

M7 $=$ VARIOVALVE

|  | - MAGNET UNDER VOLTAGE |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | FORWARD |  |  |  |  |  | REVERSE |  |  | NEUTRAL |
| GANG | 1 | 2 | 3 | 4 | 5 | 6 | 1 | , | 3 |  |
| M1 |  |  |  |  |  |  | $\bullet$ | - | $\bullet$ |  |
| M2 | $\bullet$ | - |  |  |  |  | $\bullet$ |  |  |  |
| M3 | - |  | - |  | - |  |  |  |  |  |
| M4 | $\bullet$ | - | $\bullet$ | $\bullet$ |  |  | $\bullet$ | - |  |  |
| M5 |  | - |  | - |  | - |  |  |  |  |
| clutch | KV | K4 | KV | K4 | KV | K4 | KR | KR | KR | K3 |
|  | K1 | K1 | K2 | K2 | K3 | K3 | K1 | K2 | K3 |  |



# SCHEDULE OF MEASURING POINTS 6 WG-200 WITH ALUMINUM 



DIE-CASTING SHIFT CONTROL AND WK-VALVE ${ }^{5}$ TABLE - 17

THE MEASUREMENTS HAVE TO BE CARRIED OUT AT HOT TRANSMISSION ( ABOUT. $80^{\circ}-95^{\circ} \mathrm{C}$ ) THE MARKED POSITIONS (e.g. 53) CORRESPOND WITH THE POSITIONS ON TABLE 21 !!

TRANSMISSION DIAGRAM
MESASURING POINTS FOR PRESSURE OIL AND TEMPERATURE
51 = IN FRONT OF THE CONVERTER
OPENING PRESSURE 8,5 bar
$53=$ CLUTCH FORWARD
55 = CLUTCH REVERSE
$56=$ CLUTCH
$57=$ CLUTCH
$58=$ CLUTCH
$60=$ CLUTCH
$63=$ BEHIND THE CONVERTER

|  | M10x1 |
| :--- | :--- |
| KV | M10x1 |
| KR | M10x1 |
| K1 | M10x1 |
| K2 | M10x1 |
| K3 | M10x1 |
| K4 | M10x1 |
|  | M14x1,5 |

TEMPERATURE $\left(100^{\circ} \mathrm{C}\right.$, SHORT-TIME $\left.120^{\circ} \mathrm{C}\right)$
AND
OPENING PRESSURE 2,5 bar
$65=$ SYSTEM PRESSURE $16+2$ bar
$66=$ REDUCING VALVE 10 bar
$67=$ WK CONTROL PRESSURE $13 \pm 1$ bar
MEASURING POINTS FOR DELIVERY RATES
$15=$ CONNECTION TO HEAT EXCHANGER
16 CONNECTION FROM HEAT EXCHANGER
M26x1,5

## INDUCTUVE TRANSMITTER

$1=$ INDUCTIVE TRANSMITTER TURBINE SPEED M18x1,5
$2=$ INDUCTIVE TRANSMITTER OUTPUT SPEED M18x1,5


SOLENOID VALVES
M6 = WK-VALVE


M26x1,5


M10x
M10x 1
M10x1


## SCHEDULE OF MEASURING POINTS 6 WG-200 WITH ALUMINUM DIE-CASTING SHIFT CONTROL, WK-VALVE, VARIOVALVE AND RETARDER

 TABLE - 18THE MARKED POSITIONS (e.g. 53) CORRESPOND WITH THE POSITIONS ON TABLE $22!1$ THE MEASUREMENTS HAVE TO BE CARRIED OUT AT HOT TRANSMISSION (ABOUT $80^{\circ} \mathrm{C}-95^{\circ} \mathrm{C}$ ) ।

MEASURING POINTS FOR PRESSURE OIL AND TEMPERATURE
$51=$ IN FRONT OF CONVERTER OPENING PRESSURE 8,5 bar
53 = CLUTCH FORWARD
$55=$ CLUTCH REVERSE
$56=$ CLUTCH
$57=$ CLUTCH
$58=$ CLUTCH
$60=$ CLUTCH


KR
KI
K1
K 2
K 3
K4
$63=$ BEHIND THE CONVERTER
TEMPERATURE ( $100^{\circ} \mathrm{C}$, SHORT-TIME $120^{\circ} \mathrm{C}$ ) AND OPENING PRESSURE 2,5 bar
$65=$ SYSTEM PRESSURE $16+2$ bar
$66=$ REDUCING VALVE 10 bar
$67=$ WK CONTROL PRESSURE $13 \pm 1$ bar
MEASURING POINTS FOR DELTVERY RATES
$15=$ CONNECTION TO HEAT EXCHANGER
16 = CONNECTION FROM HEAT EXCHANGER
27 = CONNECTION IN FRONT OF CONVERTER TO HEAT EXCHANGER (THROUGH RETARDER VALVE)
28 = CONNECTION BEHIND CONVERTER TO HEAT EXCHANGER (THROUGH RETARDER VALVE)
$29=$ CONNECTION FROM HEAT EXCHANGER TO LUBRICATION (THROUGH RETARDER VALVE)
$30=$ CONNECTION RETURN FLOW INTO SUMP INDUCTIVE TRANSMITTER
$1=$ INDUCTIVE TRANSMITTER TURBINE SPEED
$2=$ INDUCTIVE TRANSMITTER OUTPUT SPEED
SOLENOID VALVES
M6= WK-VALVE
M7 - VARIOVALVE

M10x1 M10xI M10x1 M10x 1 MIOxI MIOxI M10xI M14x1,5 MIOxI M10xl M10xl M45*2 M45×2 M26x1,5

M26x1,5
M26x1.5 M26x1,5 M18x1,5 $\mathrm{M} 18 \times 1,5$



## SCHEDULE OF MEASURING POINTS 6 WG-200 WITH ALUMINUM DIE-CASTING SHIFT CONTROL, WK-VALVE, AND RETARDER

 TABLE - 19THE MARKED POSITIONS (e.g. 53) CORRESPOND WITH THE POSITIONS ON TABLE $23!!$ THE MEASUREMENTS HAVE TO BE CARRIED OUT AT HOT TRANSMISSION (ABOUT $80^{\circ} \mathrm{C}-95^{\circ} \mathrm{C}$ ) !

MEASURING POINTS FOR PRESSURE OIL AND TEMPERATURE
$51=$ IN FRONT OF CONVERTER OPENING PRESSURE 8,5 bar
$53=$ CLUTCH FORWARD
M10x 1
$55=$ CLUTCH REVERSE
56 = CLUTCH
57 = CLUTCH
$58=$ CLUTCH
$60=$ CLUTCH
K1
$\begin{array}{ll}\text { CLUTCH } & \text { K2 }\end{array}$
$63=$ BEHIND THE CONVERTER TEMPERATURE ( $100^{\circ} \mathrm{C}$, SHORT-TIME $120^{\circ} \mathrm{C}$ ) AND OPENING PRESSURE 2,5 bar
$65=$ SYSTEM PRESSURE $16+2$ bar
$66=$ REDUCING VALVE 10 bar
$67=$ WK CONTROL PRESSURE 131 bar MEASURING POINTS FOR DELIVERY RATES
$15=$ CONNECTION TO HEAT EXCHANGER
$16=$ CONNECTION FROM HEAT EXCHANGER
$27=$ CONNECTION IN FRONT OF CONVERTER TO HEAT EXCHANGER (THROUGH RETARDER VALVE)
$28=$ CONNECTION BEHIND CONVERTER TO HEAT
EXCHANGER (THROUGH RETARDER VALVE)
$29=$ CONNECTION FROM HEAT EXCHANGER TO LUBRICATION M26x1,5 (THROUGH RETARDER VALVE)
$30=$ CONNECTION RETURN FLOW INTO SUMP
M26x1,5

## NDUCTIVE TRANSMITTER

$1=$ INDUCTIVE TRANSMITTER TURBINE SPEED
$2=$ INDUCTIVE TRANSMITTER OUTPUT SPEED SOLENOID VALVES
M6 = WK-VALVE



## OIL CIRCULATION DIAGRAM 6 WG-200 WITH ELECTRIC SHIFT CONTROL 2-STAGE BLEEDER VALVE; VARIOVALVE AND WK - NEUTRAL TABLE - 20

THE MARKED POSITIONS (e.g. 53 ) CORRESPOND TO THE POSITION
ON THE TABLE 16 .

## OIL CIRCULATION DIAGRAM 6 WG-200

 WITH ELECTRIC SHIFT CONTROL AND WK - N EURTAL TABLE - 21THE MARKED POSITIONS (e.g. 53) ) CORRESPOND TO THE POSITION


## OIL CIRCULATION DIAGRAM 6 WG-200 WITH ELECTRIC SHIFT CONTROL 2-STAGE BLEEDER VALVE; VARIOVALVE; WK AND RETARDER - NEUTRAL -

 TABLE - 22THE MARKED POSITIONS (e.g. 53) ) CORRESPOND TO THE POSITION 2 IABLE


## OIL CIRCULATION DIAGRAM 6 WG-200 WITH ELECTRIC SHIFT CONTROL AND WK - NEUTRAL

 TABLE - 23

## ALUMINUM DIE-CAST SHIFT CONTROL - 5-MAGNET VERSION WITH 2-STAGE; VARIOVALVE AND WK

 TABLE - 24

DOUBLE PRESSURE CONTROL VALVE WITH VARIOVALVE AND WK-VALVE (ILLUSTRATED AT A 90-TURN)


RESET VALVE
$\qquad$

## ALUMINUM DIE-CAST SHIFT CONTROL 5-MAGNET VERSION WITH WK

TABLE - 25


## 6 WG-200 POWER-SHIFT CIRCUIT DIAGRAM 6029700426 - SAMPLE - DUMPER „CODE H" <br> TABLE - 26



## NOTE:

BINDING IS THE CORRESPONDING SPARE-PARTS LIST VERSION
CONNECTION DIAGRAM - 6029700428 - SEE TABLE 27

## 6 WG-200 POWER SHIFT - CONNECTION DIAGRAM - 6029700428 -

 SAMPLE - DUMPER "CODE H" TABLE - 27

ILLUSTRATED POLE PATTERNS
VIEW UPON PLUG-IN SIDE OF THE INSTRUMENTS



AMP 8-POLE PIN


AMP 8-POLE PIN


CANNON 6-POLE CA PIN


CIRCULAR PLUG
2-POLE K24

POLE PATTERN Y6 POLE PATTERN A2-X1


## NOTE:

- BINDING IS THE CORRESPONDING SPARE-PARTS

| POS | LEGEND |
| :---: | :--- |
| A2 | CONTROLLER |
| A3 | ELEKTRO-HYDR. CONTROL WG |
| A4 | WK-MODULE |
| B2 | SPEED SENSOR TURBINE |
| F1 | FUSE 8A |
| K1 | RELAY STARTER INTERLOCK |
| K2 | RELAY BACKUP LAMP |
| Y6 | CONVERTER LOCKUP CLUTCH |


| SUPPLY SYSTEM, VEHICLE INTERFACE |  |  |
| :---: | :---: | :---: |
| 1 |  | CURRENT SUPPLY |
| 2 |  |  |
| 3 |  | GROUND |
| 4 |  | SM-SIGNAL BLOCKS DOWNSHIFTING ON |
|  |  | GEARSHIFT LEVER SPEED-DEPENDENT |
| 5 |  | CURRENT SUPPLY |
| 6 |  | SM-SIGNAL BLOCKS DOWNSHIFTING ON |
|  |  | GEARSHIFT LEVER IN "R" POSITION W-SIGNAL SWITCHES LOCKUP CLUTCH |
| 7 |  | OFF AND PUTS RETARDER OUT OF |
|  |  | SERVICE IF RETARDER SWITCH S1 IS ACTIVATED |
| 8 |  | IF GEARSHIFT LEVER IN "N" POSITION |

VERSION CIRCUIT DIAGRAM - 6029700426 - SEE TABLE 26 -

6 WG-200 POWER-SHIFT CONNECTION DIAGRAM - 6029700616 -- SAMPLECRANE VEHICLE "CODE H" -


## 6 WG-200 SEMI- AND FULLY-AUTOMATIC CONTROL

CIRCUIT DIAGRAM - 6029700800 -
SAMPLE DUMPER „CODE H ,
TABLE - 29


ILLUSTRATED POLE PATTERNS
VIEW UPON PLUG-IN SIDE OF INSTRUMENTS


POLEPATTERNB:


CIRCULAR PLJG
2 -FOLES24

POLEPATTERN A2.X2, W1-XI


AMP 8-POLEPIN


CANEOON GPOL CA PRN

## POLEPATIERNA2 - XI



NMP 8-POLE SOCKEI

## NOTE:

- BINDING IS THE CORRESPONDINO PARTS-LIST VERSION

| POS | LEGEND |
| :---: | :---: |
| A1 | CONTROLLER LOWER CHASSIS |
| A2 | CONTROLLER SUPERSTRUCTURE |
| A3 | ELECTRO-HYDR. CONTROL WG |
| A4 | WK-MODULE |
| B2 | SPERD SENSOR TURBINE |
| F1 | FUSE SA |
| K1 | RELAY STARTER INTERLOCK |
| K 2 | RELAY BACKUP LAMP |
| K3 | RELAY RETARDER |
| K4 | RELAY FRONT AXIE |
| S1 | SWITCH RETARDER |
| \$2 | SWITCH PRESSURE CUT-OFF <br> (ONLY AFTER AGREEMENT WITH ZF) |
| \$3 | SWITCH LOWER CHASSIS |
| S4 | SWITCH SUPERSTRUCTURE |
| S6 | SWITCH IDLE ACCELERATOR (CLOSED IN IDLE RUN) |
| S7 | SWITCH PRESSURE (WK) |
| S8 | SWITCH FRONT AXLE |
| Y6 | CONVERTER LOCKUP CLUTCH |
| Y7 | RETARDER |


| SUPPLY SYSTEM, VEHICLE INTERFACE |  |  |
| :---: | :---: | :---: |
| 1 | 1 | POWER SUPPLY |
| 2 |  |  |
| 3 | 1 | OROUND |
| 4 | 1 | SM-SIGNAL BLOCKS DOWNSHIFTING ON GBARSHIFT LEVER SPERD-DEPENDENTLY |
| 5 |  |  |
| 6 | - | SM-SIGNAL BLOCKS DOWNSHIPTING ON GEARSHIFT LEVER IN 'R" POSITION |
| 7 | - | W-SIONAL SWITCHES LOCKUPCLUTCH OFF AND PUTS THE RETARDER OUT OF SERVICE IF RETARDER SWITCHSI IS ACTIVATED |
| 8 | - | RELAY STARTER INTERLOCK ACTIVATED IF GEARSHIFT LEVER IN "N" POSITION |

## 6 WG-200 SEMI- AND FULLY-AUTOMATIC CONTROL CONNECTION DIAGRAM - 6029700789 -

 SAMPLE - DUMPER "CODE H" TABLE - 30

- BINDINO IS THE CORRESPONDING PARTS-LIST

VERSION - CONNECTION DLAGRAM - 6029700800 - SEE TABLE 30.

## 6 WG-200 SEMI- AND FULLY-AUTOMATIC CONTROL CIRCUIT DIAGRAM - 6029700828 SAMPLE CRANE VEHICLE "CODE H"

 TABLE - 31

## 6 WG-200 SEMI- AND FULLY-AUTOMATIC CONTROL CONNECTION DIAGRAM - 6029700836 SAMPLE - CRAN VEHICLE , CODE H ,

 TABLE - 32

ILLUSTRATED POLE PATTERNS
VIEW ON PLUG-IN SIDE OF THE INSTRUMENTS


POLE PATTERN =W1-X4


NOTE:

- BINDING IS THE CORRESPONDING PARTS-LIST

VERSION CIRCUIT DIAGRAM - 6029700828 - SEE TABLE 31.

## 6 WG-200 SEMI- AND FULLY-AUTOMATIC CONTROL CIRCUIT DIAGRAM CONTROLLER - 6006700197 -

 SAMPLETABLE - 33


## INDUCTIVE TRANSMITTER AND ELECTRONIC MODULE

TABLE - 34

## INDUCTIVE TRANSMITTER

The Inductive transmitter is recording the turbine speed for the control of the converter clutch and change-back lock.
It is fitted accessibly on the transmission and is operating without mechanical wear.
The transmitter is connected with the Electronic Module through a cable with 3-pole bayonet-plug.

For the recording of the output speed at the speeddependently operating automatic shift control as well as for the electric speedometer, an inductive transmitter is also used.

## ELECTRONIC MODULE

The Electronic Module is converting the pulses issued by the inductive transmitter into switching points for the control of the converter clutch and the change-back lock.
These switching points are determined according to the type of vehicle.
The Electronic Module is of the splash-protected design and is connected through a 8 -pole AMP-Flat-type connector.


ELECTRONIC MODULE

- For the Test of electric transmission control units with 4 and 5 solenoid valves .

For the Test of the Controllers SG-4; SG-6 and SG-6s in and out of the vehicle.

Especially suitable for the Version with 8 -pole AMP Supply system wiring .

Not suitable for the Test of electronic transmissions control units , as : EST - 2 /EST - 17 /EST - 17 T/EST - 19 /EST - 25 /EST - 26

Monchaviz
CONTROLLER


To be used combined with : Digital - Multimeter 5870221129

Modifications reserved !

The conception of the Tester generation PR-78 combines a high degree of flexibility with handiness and a universal range of applications.
It allows the testing of electric and electronic gearbox control units, developped by ZF.
EST-2 / EST-17 / EST-17 T/EST-19 T/EST-25 / EST-26
Controller DW-2 / SG-4 / SG-6s / SG-8s
Transmission with 4,5 and 6 Solenoid valves
It allows the testing of electric and electronic gearbox control units, developped by ZF,


Because of the interchangeability of the connecting cables and Project Foils, the Tester can be adapted to the acutal check problem.

Adapter cables are delivered together with the Test specifications.
Ancillary Instruments can be directly connected through the Testes PR-78 B, thus enlarging possibilities, as for example!

## CURRENT - VOLTAGE - RESISTANCE - FREQUENCY

The Tester PR-78 A is equipped with a 22 LED display and represents therefore the idieal Instrument for the system and functional test.

The T-Adapter 6008201013 allows the combination of the Tester PR-78 A and PR-78 B to form a complete test unit.

The test can be carried out in the vehicle or in removed condition.

Table 37


## SUMMARY

 PR-78
## CABLE



TESTER PR - 78 A TESTER PR - 78 B are used together with T-adapter cable 6008201013



Test the output signals by 4 and 5 Solenoids together with some measuring devices

## CURRENT - VOLTAGE - RESISTANCE - FREQUENCY

To be used together with Tester PR - 78 B 6008001002 and foil 6008301029 and T-Adaptercable 6008201013

Zusaitzlich in Verbindung mit Multimeter 5870221103 oder 5870221129

Table 39

The Test - System PR - 78 A and PR - 78 B allows the testing of electronic gearboxcontrol EST - 2 /EST - 17 T/EST - 17 TE/EST - 19 T/EST - $25 /$ EST - 25 E/EST - 26
In combination with different gearselectors like :
SG-6/SG-6S / SG-8S / DW-2/ ERGO-II / push-button switches D-7 /D-9

Tester PR - 78 A with LED displays
Order- No. 6008001001


Foil 6008301015


T - Adapter cable 6008201013 allows the combination of Tester PR - 78 A and PR- 78 B to form a complete test unit.


Ancillary Instruments can be directly connected through the Tester PR-78 B, thus enlarging possibilities as for example : CURRENT - VOLTAGE - RESISTANCE - FREQUENCY
Digital-Multimeter $\quad 5870221129$ or 5870221103
Kableset 2x 5870221014

TESTER

The Tester PR-68 allows the testing of electronic control units, developed by ZF.
EST - $2 /$ EST - $17 /$ EST - 17 T/EST - 19 T/EST - 25/EST - 26
Transmissions with 4-5- and 6 Solenoid valves.


A major improvement of the tester PR - 68 is the compact construction as also integrated frequenzgenerator. With the frequenz - generator it is possible to simulate the turbine- and output signal . Therefore a function test by vehicle stop can done !
NOTE :
There are two versions of tester PR - 68 aviable .

1: Standard version
2: Rugged version
$5870221045 \quad 35-$ pol. connection AMP $5870221079 \quad 37$-pol. connection Cannon


Handy Measuring instrument for the mobile use. LCD - display with high - contrast 7- Segment and Bargraph display.
for measurings as:
CURRENT - VOLTAGE - RESISTANCE - FREQUENCY
capacitance measurement -

- Transistortest - Auto - Hold - Function .
- Min. / Max . - Average - value memory .


This Instrument is delivered complete with protective rubber cap and measuring cords .
Replacement for 5870221103
Modifications reserved !


 Thamal mirtion- paper


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vacimmosion imerimaze




Sotiwave suppart throgh Hitredcumps Poasuring vave aedeuston $181012125 / \mathrm{E}$

Escernal powet siepty 34 VDC


Vhicile sotrecton cate karl 229 254

${ }^{*}$ \&-Combinutic) Sonlinypole
 2spotenpole

Multi - System 5000 (new) Order - No. 5870221250
4 analog channels for standardiznd input signais, which can be defined freely: 0 to 20 mA and 4 to 20 mA , 4.9 .

Automatical tanster of massuring values into helptal graphics




2 digital channels as frequency (puises), which can be defined freely, e.g. volume fow reby Hein th. galimin!




[^0]:    * = According to the Transmission Version, also other ratios are possible.

[^1]:    M26x1,5

